

DISPLACEMENT CYLINDERS - INDIRECT ACTING

PUMP LPM	CYLINDER TYPE									LOADED STATIC PRESSURE														
	30	40	50	57	63	70	80	90		27	29	31	33	35	37	39	41	43	45	47	49	51	53	
8	0.32	0.2	0.14	0.1	0.09	0.07																		
15	0.64	0.4	0.24	0.2	0.16	0.14	0.1	0.08																
23	0.9	0.6	0.38	0.3	0.24	0.2	0.14	0.12																
35		0.9	0.56	0.44	0.36	0.3	0.22	0.18																
50			0.8	0.62	0.5	0.42	0.32	0.24																
75			1.2	0.96	0.76	0.64	0.48	0.38																
100				1.28	1.02	0.84	0.64	0.5																
125						1.06	0.8	0.64																
150							0.98	0.76																
180							1.18	0.92																
210								1.08																
250																								
300																								
320																								
380																								
440																								
500																								
650																								
800																								

-- SPEED OBTAINED (ms⁻¹) --

-- MOTOR POWER REQUIRED --

Use the buckling graph to determine a suitable **cylinder type** to lift the car/sling & load. This graph will also dictate the **loaded static pressure**.

For Example:

The buckling graphs indicate that the most suitable cylinder is a **80** and the loaded static pressure is **43 bar**.

I want the lift to move at approximately **0.8 m/s**.

- (1) Locate the **cylinder type** first - **80**.
- (2) Move vertically down the column to the nearest speed shown to what is required, I want 0.8 m/s so the nearest available would be **0.8 m/s**.
- (3) The **pump** size is determined by moving horizontally left across the row to the far left column, which gives a **pump** size of **125 lpm**.
- (4) The motor size is determined by moving horizontally right across the row until you are vertically below the **loaded static pressure** of the lift - **43 bar**, which gives a **motor** size of **16kW**.

DISPLACEMENT CYLINDERS - INDIRECT ACTING

PUMP LPM	CYLINDER TYPE									LOADED STATIC PRESSURE															
	100	110	125	140	160	180	200	220		27	29	31	33	35	37	39	41	43	45	47	49	51	53		
8																									
15																									
23	0.09	0.08																							
35	0.14	0.12	0.09																						
50	0.2	0.16	0.13	0.1	0.08																				
75	0.3	0.26	0.2	0.16	0.12	0.1																			
100	0.42	0.34	0.26	0.22	0.16	0.13	0.1																		
125	0.52	0.42	0.34	0.26	0.2	0.16	0.13	0.11																	
150	0.62	0.52	0.4	0.32	0.24	0.2	0.16	0.13																	
180	0.74	0.62	0.48	0.38	0.3	0.24	0.18	0.15																	
210	0.88	0.72	0.56	0.44	0.34	0.28	0.22	0.18																	
250	1.04	0.86	0.66	0.54	0.4	0.32	0.26	0.22																	
300		1.04	0.8	0.64	0.48	0.38	0.32	0.26																	
320			0.86	0.68	0.52	0.42	0.34	0.28																	
380			1.02	0.8	0.62	0.5	0.4	0.32																	
440				0.94	0.72	0.58	0.46	0.38																	
500				1.06	0.82	0.64	0.52	0.44																	
650					1.06	0.84	0.68	0.56																	
800						1.04	0.84	0.7																	

-- SPEED OBTAINED (ms⁻¹) --

-- MOTOR POWER REQUIRED --

Use the buckling graph to determine a suitable **cylinder type** to lift the car/sling & load. This graph will also dictate the **loaded static pressure**.

For Example:

The buckling graphs indicate that the most suitable cylinder is a **110** and the loaded static pressure is **39 bar**.

I want the lift to move at approximately **0.5 m/s**.

- (1) Locate the **cylinder type** first - **110**.
- (2) Move vertically down the column to the nearest speed shown to what is required, I want 0.5 m/s so the nearest available would be **0.52 m/s**.
- (3) The **pump** size is determined by moving horizontally left across the row to the far left column, which gives a **pump** size of **150 lpm**.
- (4) The motor size is determined by moving horizontally right across the row until you are vertically below the **loaded static pressure** of the lift - **39 bar**, which gives a **motor** size of **16kW**.